Application of anisotropic Hyperelastic constitutive model in finite element analysis of flexible deformable bump

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Abstract

The three-dimensional nonlinear bump model is established by using ABAQUS software. The anisotropic hyperelastic model established by the strain energy density function , which is used to simulate the property of cord - rubber composites in the flexible bump. The main direction of the material can simulate the geometrical and material nonlinearity of the cord - rubber composite. By modeling the bump structure as a whole, the model is simplified and the calculation efficiency and calculation accuracy are improved. The deformation height and inner stress of the bump structure under different inflation pressures are obtained by three-dimensional nonlinear analysis, the large deformation of cord - rubber composite is also taken into account in the analysis.

Keywords: bump; anisotropic hyperelastic; cord - rubber composites; nonlinear analysis; deformation

Introduction

The cord - rubber composite is a flexible composite material formed by the effective combination of the cord and the rubber. It is widely used in tires, inflatable springs, air bags and conveyors. The rubber is a material with a low elastic modulus and a high elongation, while the cord is a material with a high modulus of elasticity and a low elongation. The composite exhibits a high toughness and a high strength by the cooperation of the substrate and the fibers high strength characteristics. Because of the obvious anisotropy and nonlinearity of the cord rubber material during the bearing, the anisotropic hyperelastic constitutive model[1]-[4] can well characterize the characteristics of the cord - rubber composite , and has good engineering practicability

composite, and has good engineering practicability In this paper, ABAQUS is used to establish the finite element model of the bump. By the UANISOHYPER_INV interface in the ABAQUS do secondary development of the constitutive model, called the written UANISOHYPER_INV subroutine as cord - rubber composite constitutive equations during the finite element calculation. This can more accurately simulate the deformation behavior of the cord - rubber composite structure.

1. Anisotropic hyperelastic constitutive model

Anisotropic elastic constitutive model is based on the continuum mechanics of fiber reinforced composite material[5]-[7]. the fiber composite material having a direction, the strain energy function can be expressed as the function of the right Cauchy-Green deformation tensor $\mathbf{C} = \mathbf{F}^T \mathbf{F}$ and the invariant I_i which is related to the original fiber directional vector \mathbf{a}_0 . Generally have the following form, Here \mathbf{F} is the deformation gradient tensor

$$W(\mathbf{C}, \mathbf{a}_{0}) = W_{iso}(I_{1}, I_{2}, I_{3}) + W_{triso}(I_{4})$$
 1-1

Where W_{iso} is the isotropic hyperelasticity of the the rubber, W_{triso} is the anisotropic strain energy due to the reinforcement of the cord; the strain tensor invariant I_1, I_2, I_3 characterizes

the isotropic property of the material, which I_4 is related to the direction of the material. Where the invariants are given by

$$I_{1} = tr(C); I_{2} = \frac{1}{2} [(trC)^{2} - tr(C)^{2}]$$

$$I_{3} = det(C); I_{4} = a_{0} \cdot C \cdot a_{0}$$

1-2

Where $\mathbf{c} = \begin{bmatrix} \lambda_1^2 & 0 & 0 \\ 0 & \lambda_2^2 & 0 \\ 0 & 0 & \lambda_3^2 \end{bmatrix}$, λ_i represents a ratio of three principal directions of elongation

material ($\lambda_i = 1 + \varepsilon_i, \varepsilon_i$ represents the strain of principal directions).

In an ideal conditions, the rubber can be regarded as incompressible material, The energy W_{iso} stored in the matrix is given by the Mooney-Rivlin model[8]-[10] form

$$W_{\rm iso} = \sum_{ij=0}^{ij=n} C_{ij} (l_1 - 3)^i (l_2 - 3)^j$$

The Mooney-Rivlin two-parameter strain energy function, which is widely used in ABAQUS finite element software, can be obtained by the first two terms of the equation in the case of sufficiently small deformation

 $W_{iso} = C_{10}(I_1 - 3) + C_{01}(I_2 - 3)$

Where C_{10} , C_{10} are the material parameters, the unit is Mpa.

The strain energy of the cord is generally considered to be related to the elongation of the cord and equal to the square of the cord draw ratio. Therefore, a polynomial is defined to represent the strain energy of the cord. When the cord is compressed, Can be ignored. The strain energy function of the cord is as follows

$$W_{triso} = \begin{cases} k_1 (I_4 - 1)^2 + k_2 (I_4 - 1)^3 + k_3 (I_4 - 1)^4, I_4 > 1\\ 0 \end{cases}$$

Where $k_1 \ k_2 \ k_3$ are material parameters, in units of Mpa.

In order to obtain a stress-strain constitutive relation of the form, according to the formula 1-1 derived composite material the second Piola- Kirchhoff stress tensor S is

$$\mathbf{S} = 2 \frac{\partial W(\mathbf{C}, \mathbf{a}_0)}{\partial \mathbf{C}} = 2 \sum_{m=1}^{4} \frac{\partial W(\mathbf{C}, \mathbf{a}_0)}{\partial I_m} \frac{\partial I_m}{\partial \mathbf{C}}$$

Where the invariant I_m pairs of right Cauchy - Green deformation tensor partial derivatives are as follows:

$$\frac{\partial I_1}{\partial \mathbf{C}} = \mathbf{I} \qquad \qquad \frac{\partial I_2}{\partial \mathbf{C}} = I_1 \mathbf{I} - \mathbf{C}$$
$$\frac{\partial I_3}{\partial \mathbf{C}} = I_2 \mathbf{I} - I_1 \mathbf{C} + \mathbf{C}^2 \qquad \frac{\partial I_4}{\partial \mathbf{C}} = \mathbf{a}_0 \otimes \mathbf{a}_0$$

Cauchy stress tensor

$$\boldsymbol{\sigma} = \boldsymbol{J}^{-1} \mathbf{F} \mathbf{S} \mathbf{F}^{T} = \frac{2}{J} \Big[(W_1 + W_2 I_1 + W_3 I_2) \mathbf{B} - (W_2 + W_3 I_1) \mathbf{B}^2 + W_3 \mathbf{B}^3 + I_4 W_4 \mathbf{a} \otimes \mathbf{a} \Big]$$

Which W_i represents the partial derivative of the invariant, **B** represents the left Cauchy-Green deformation tensor, and the post-deformation cord direction $\mathbf{a} = \frac{1}{\lambda_F} \mathbf{F} \mathbf{a}_0$, (λ_F representing the elongation ratio of the cord)

$$\begin{cases} \sigma_{1} = 2 \Big[(W_{1} + W_{2}I_{1})\lambda_{1}^{2} - W_{2}\lambda_{1}^{4} + W_{4}\lambda_{1}^{2}\cos^{2}\alpha \Big] \\ \sigma_{2} = 2 \Big[(W_{1} + W_{2}I_{1})\lambda_{2}^{2} - W_{2}\lambda_{2}^{4} + W_{4}\lambda_{2}^{2}\sin^{2}\alpha \Big] \\ \sigma_{3} = 2 \Big[(W_{1} + W_{2}I_{1})\lambda_{3}^{2} - W_{2}\lambda_{3}^{4} \Big] \end{cases}$$

The material parameters in the constitutive model can be fitted by uniaxial tensile experiments (1) Get the rubber matrix material parameters C_{10} and C_{01} according rubber uniaxial stretching test data

(2) Get the cord material parameters $k_1 \, \cdot \, k_2$ and k_3 according uniaxially stretched in the direction of the cord fitting experimental data

The tensile test data is fitted by the least squares method to obtain the material parameters in the constitutive model as shown in the following table

Table 1.Material parameters

C_{10}	C_{01}	k_1	<i>k</i> ₂	<i>k</i> ₃
-0.89	1.78	13.43	-17.88	14.45

2. Three-dimensional finite element model of bump

2.1 Three-dimensional structure of the bump

The actual geometry of bump is complicated, get the airbag outside the cavity when set up the finite element model, as show in the figure(a) . the airbag on both sides of the metal chamber is determined by applying a fixed boundary conditions . the airbag section around the central axis of rotation to get bump geometry model.



Figure(a)

2.2 Mesh and element type

The bump structure is divided into three-layer. And the bump are constructed by different material layers. The unit of the model is C3D8, a total of 3840 units 6412 nodes, as show in the figure(b).



Figure(b)

2.3 Grid division and cell type

Due to the use of cord - rubber composites in the drum structure, the drum structure is divided into three-layer units when the meshes are divided, and the drums are constructed by different material layers. Three-dimensional model using 8- node entity unit, a total of 3840 units 6412 nodes.

2.4 Material model

The rubber layer use the Mooney-Rivlin material model, cord - rubber composite material layer using an anisotropic elastic constitutive material models, When defining the cord - rubber composite layer, the direction of the material can be assigned according to the actual laying angle of the cord.

2.5 Boundary conditions and load conditions

Considered the bump and the metal cavity fixed constraint, set the completely fixed constraint of two sides of bump, regard the inflation pressure as a uniform load for the inner surface of the bump in the simulation analysis.

3. Calculation result and analysis

Table 2 is the maximum deformation height of bump structure under the different loads and different cord angle, figure (c) is a maximum deformation height of bump structure under different load and the same cord angle; figure(d) is the maximum deformation height of bump structure under different cord angle and the same load.

load/ Pa angle	0.1	0.3	0.5	0.7	0.9
15 [°]	83.09	84.42	85.78	87.18	88.60
30°	83.21	84.77	86.36	88.01	89.70
45 [°]	83.43	85.38	87.40	89.55	91.77
60°	83.72	86.19	88.82	91.78	94.72

Table 2. the maximum deformation height of bump



Figure (d)

It can be seen from figure(c) that the maximum deformation height of the bump increases by the increase of the uniform load in the bump inner surface at the same cord angle, and the maximum deformation height of the bump under different loads is approximately in a straight line, It can be shown that the maximum deformation height of the bump increases linearly with the increase of the uniform load when the cord angle is unchanged. As can be seen from figure(d), with the uniform distribution load, The maximum deformation height of the bump is also increased by the increase of the cord angle, the maximum deformation height of the bump is approximately in a quadratic curve. In figure(c) and figure(d), the curve function expression acquired by the least squares method fit.

4.Conclusions

The cord-rubber composite material is a nonlinear material. The anisotropic hyperelastic constitutive model can simulate the deformation behavior of the cord-rubber accurately, and provide the analytical basis for the design and optimization of the bump structure in the subsequent finite element analysis.

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